

# Turbulent Differentially Heated Cavity of Aspect Ratio 5

## DNS and regularization modeling

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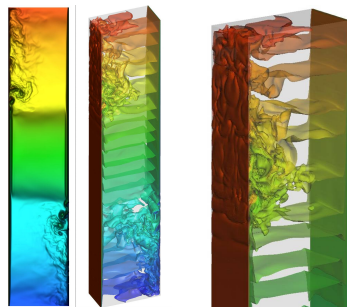
# Contents

- 1 DNS
- 2 Regularization models
- 3 Stopping the vortex-stretching
- 4 Results for a DHC
- 5 Conclusions

# DNS of turbulent incompressible flows

Main features of the DNS code:

- Structured staggered grids
- High-order symmetry-preserving schemes
- Fully-explicit second-order time-integration method
- Poisson solver for 2.5D problems: FFT + PCG
- Hybrid MPI+OpenMP parallelization

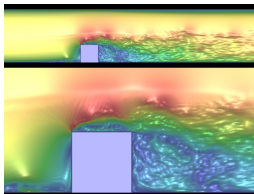


Air-filled differentially heated cavity at  $Ra = 10^{11}$  (111M grid points)

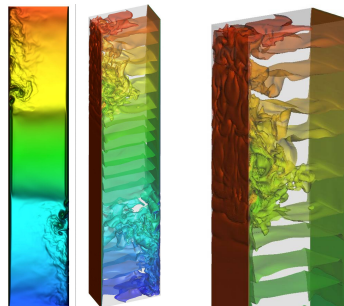


Plane impingement jet at  $Re = 20000$  (102M grid points)

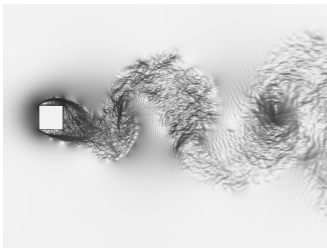
# DNS of turbulent incompressible flows



Wall-mounted cube at  $Re = 7240$  (17M grid points)



Air-filled differentially heated cavity at  $Ra = 10^{11}$  (111M grid points)

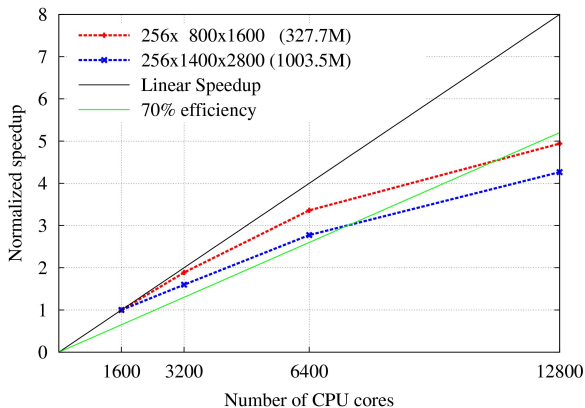


Square cylinder at  $Re = 22000$  (75M grid points)



Plane impingement jet at  $Re = 20000$  (102M grid points)

# Scaling? Yes<sup>1</sup>, we can... but never enough



<sup>1</sup>A. Gorobets et al. "Hybrid MPI+OpenMP parallelization of an FFT-based 3D Poisson solver with one periodic direction" **Computers&Fluids**, (available online).

# Governing equations

Incompressible Navier-Stokes equations:

$$\begin{aligned}\nabla \cdot u &= 0 \\ \partial_t u + \mathcal{C}(u, u) &= \mathcal{D}(u) - \nabla p\end{aligned}$$

where the **nonlinear convective term** is given by

$$\mathcal{C}(u, \phi) = (u \cdot \nabla)\phi$$

and the linear dissipative term is given by

$$\mathcal{D}(\phi) = \nu \Delta \phi$$

## Regularization modeling

As the full energy spectrum cannot be computed, a **dynamically less complex mathematical formulation** is sought. We consider smooth approximations (regularizations) of the nonlinearity,

$$\partial_t u_\epsilon + \tilde{\mathcal{C}}(u_\epsilon, u_\epsilon) = \mathcal{D}(u_\epsilon) - \nabla p_\epsilon$$

such approximations may fall in the **Large-Eddy Simulation** (LES) concept,

$$\partial_t \bar{u}_\epsilon + \mathcal{C}(\bar{u}_\epsilon, \bar{u}_\epsilon) = \mathcal{D}(\bar{u}_\epsilon) - \nabla \bar{p}_\epsilon + \mathcal{M}_1(\bar{u}_\epsilon, \bar{u}_\epsilon)$$

if the filter is invertible:

$$\mathcal{M}_1(\bar{u}_\epsilon, \bar{u}_\epsilon) = \mathcal{C}(\bar{u}_\epsilon, \bar{u}_\epsilon) - \overline{\tilde{\mathcal{C}}(u_\epsilon, u_\epsilon)}$$

## Symmetry-preserving regularization models (1/2)

In order to conserve the following inviscid invariants

- Kinetic energy :  $(u, u)$
- Enstrophy (in 2D) :  $(\omega, \omega)$
- Helicity (in 3D) :  $(\omega, u)$

where  $(a, b) = \int_{\Omega} a \cdot b d\Omega$  and  $\omega = \nabla \times u$ ; the **approximate convective operator** must be **skew-symmetric**:

$$(\tilde{\mathcal{C}}(u, \phi_1), \phi_2) = -(\tilde{\mathcal{C}}(u, \phi_2), \phi_1)$$



## Symmetry-preserving regularization models (2/2)

This criterion yields the following class of approximations<sup>2</sup>,

$$\partial_t u_\epsilon + \mathcal{C}_n(u_\epsilon, u_\epsilon) = \mathcal{D}(u_\epsilon) - \nabla p_\epsilon$$

in which the convective term is smoothed according to:

$$\mathcal{C}_2(u, \phi) = \overline{\mathcal{C}(\bar{u}, \bar{\phi})}$$

$$\mathcal{C}_4(u, \phi) = \mathcal{C}(\bar{u}, \bar{\phi}) + \overline{\mathcal{C}(\bar{u}, \phi')} + \overline{\mathcal{C}(u', \bar{\phi})}$$

$$\mathcal{C}_6(u, \phi) = \mathcal{C}(\bar{u}, \bar{\phi}) + \mathcal{C}(\bar{u}, \phi') + \mathcal{C}(u', \bar{\phi}) + \overline{\mathcal{C}(u', \phi')}$$

where  $u' = u - \bar{u}$  and  $\mathcal{C}_n(u, \phi) = \mathcal{C}(u, \phi) + \mathcal{O}(\epsilon^n)$  for **any symmetric filter**.

**High-frequencies need to be effectively damped.**

**But how much?**

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<sup>2</sup>Roel Verstappen, **Computers & Fluids**, 37 (7): 887-897, 2008

## Stopping the vortex-stretching<sup>3</sup>

Taking the curl of momentum equation the vorticity transport equation follows

$$\partial_t \omega + \mathcal{C}(u, \omega) = \mathcal{C}(\omega, u) + \mathcal{D}(\omega)$$

Let us now consider an arbitrary part of the flow domain,  $\Omega$ , with periodic boundary conditions. Then, taking the  $L^2$  innerproduct with  $\omega = \nabla \times u$  leads to the enstrophy equation

$$\frac{1}{2} \frac{d}{dt} (\omega, \omega) = (\omega, \mathcal{C}(\omega, u)) - \nu (\nabla \omega, \nabla \omega)$$

where  $(a, b) = \int_{\Omega} a \cdot b d\Omega$ . Unless, the grid is fine enough convection dominates diffusion

$$(\omega, \mathcal{C}(\omega, u)) > \nu (\nabla \omega, \nabla \omega)$$

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<sup>3</sup>F.X. Trias *et al.* **Computers&Fluids**, 39:1815-1831, 2010

## Stopping the vortex-stretching

The vortex-stretching term can be expressed in terms of the invariant

$$r = -1/3\text{tr}(S^3)$$

$$(\omega, \mathcal{C}(\omega, u)) = 4 \int_{\Omega} r d\Omega \quad (1)$$

whereas the  $L^2(\Omega)$ -norm of  $\omega$  in terms of the invariant  $q = -1/2\text{tr}(S^2)$

$$(\omega, \omega) = -4 \int_{\Omega} q d\Omega$$

Then, the diffusive term can be bounded by

$$\nu(\nabla\omega, \nabla\omega) = -\nu(\omega, \Delta\omega) \leq -\nu\lambda_{\Delta}(\omega, \omega) = 4\nu\lambda_{\Delta} \int_{\Omega} q d\Omega \quad (2)$$

where  $\lambda_{\Delta} < 0$  is the largest (smallest in absolute value) non-zero eigenvalue of Laplacian operator  $\Delta$  on  $\Omega$ . If we now consider that the domain is a periodic box of volume  $h$ , then  $\lambda_{\Delta} = -(\pi/h)^2$ .

## Stopping the vortex stretching

⇒ In the present work we **determine the filter width**  $\epsilon$  from

$$(\omega, \mathcal{C}_4(\omega, u)) \approx f_4(\hat{g}_k(\epsilon))(\omega, \mathcal{C}(\omega, u)) \leq \nu(\nabla\omega, \nabla\omega)$$

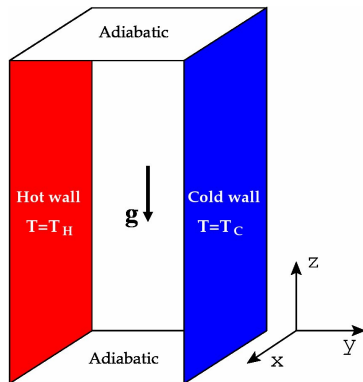
Then, recalling identity (1) and inequality (2), we propose to rewrite the previous inequality in terms of the invariants  $q$  and  $r$

$$f_4(\hat{g}_k) = \min \left\{ \nu \lambda_\Delta \frac{q}{r^+}, 1 \right\} \quad \text{with } r^+ = \max(r, 0)$$

Notice that  $q < 0$  (**dissipation**) whereas  $r$  can be either positive or negative.

- Switches off ( $f_4 = 1$ ) for: laminar ( $r \rightarrow 0$ ), 2D flows ( $r = 0$ ) and for fine enough meshes,  $|\nu \lambda_\Delta q/r| \geq 1$ .
- Consistent near-wall behavior  $r \propto y^3$  and  $q \propto y^0$ .
- Consistent with the preferential vorticity alignment with the intermediate eigenvector,  $\lambda_2$  (experimentally observed)

# Test-case: Differentially Heated Cavity



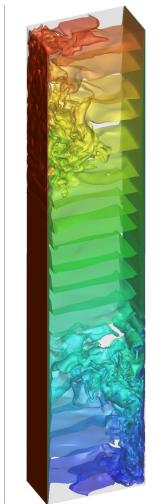
Boundary conditions:

- **Isothermal vertical walls**
- **Adiabatic horizontal walls**
- **Periodic** boundary conditions in the spanwise direction

Dimensionless governing numbers:

- $Ra = \beta \Delta T L_z^3 g / (\alpha \nu)$
- $Pr = \nu / \alpha$
- Height aspect ratio  $A_z = L_z / L_y$
- Depth aspect ratio  $A_x = L_x / L_y$

# DNS results for $Ra = 4.5 \times 10^{10}$ , $Pr = 0.7$

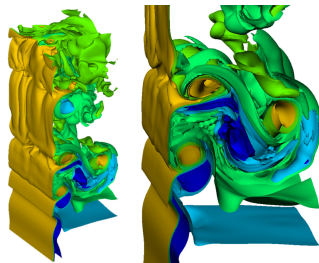


## Some details about **DNS**:

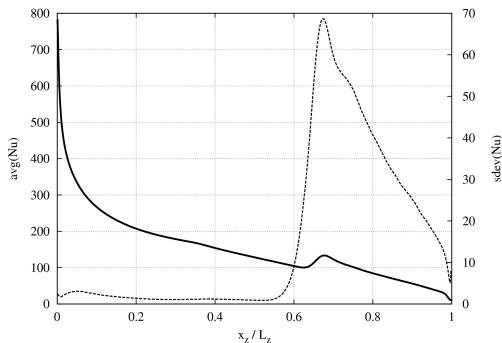
- Mesh size:  $128 \times 318 \times 862$
- $\approx 1$  months - 128 CPUs
- 4<sup>th</sup>-order symmetry-preserving scheme
- $A_z = 5$

## Complexity of the flow:

- Boundary layers
- Stratified cavity core
- Internal waves
- Recirculation areas



# DNS results for $Ra = 4.5 \times 10^{10}$ , $Pr = 0.7$



Transition point is located at  $z/L_z \approx \mathbf{0.67}$  whereas experiments (also some LES and RANS simulations) predicted much more upstream positions ( $z/L_z \approx \mathbf{0.2}$ ). This discrepant behavior was also observed for a DHC<sup>4,5</sup> of  $A_z = 4$ ,  $Ra \leq 10^{11}$ .

<sup>4</sup>F. X. Trias *et al.* Int. Journal of Heat and Mass Transfer, 53:665-673, 2010

<sup>5</sup>F. X. Trias *et al.* Int. Journal of Heat and Mass Transfer, 53:674-683, 2010

# Results for differentially heated cavity at $Ra = 4.5 \times 10^{10}$

- Regularization model  $\mathcal{C}_4$  is tested.
- Two coarse meshes are considered

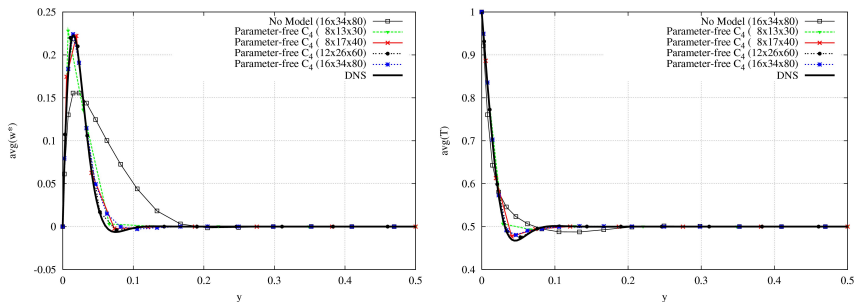
	DNS	RM1		RM2	
Mesh	$128 \times 318 \times 862$	$8 \times 20 \times 54$		$8 \times 14 \times 38$	
		No model	$\mathcal{C}_4$	No model	$\mathcal{C}_4$
$Nu$	154.5	194.1	<b>157.5</b>	210.5	<b>159.4</b>
$Nu_{max}$	781.5	535.2	682.6	558.2	711.8
$Nu_{min}$	10.5	86.9	18.1	93.7	15.1

- The **discrete linear filter**<sup>6</sup> is based on polynomial functions of the discrete diffusive operator,  $\mathbf{D}$

<sup>6</sup>F.X. Trias and R.W.C.P. Verstappen, **Computers & Fluids**, 40:139-148, 2011



# How does the parameter-free $\tilde{C}_4$ regularization modeling behave for other grids and $Ra$ -numbers?

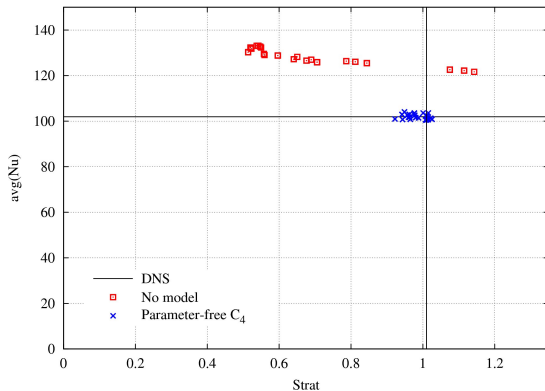


Averaged vertical velocity and temperature profiles at the horizontal mid-height plane at  $Ra = 10^{10}$ .

Even for a **very coarse**  $8 \times 13 \times 30$  grid **reasonable results** are obtained!

$\implies$  Results for different grids show the **robustness** of the method.

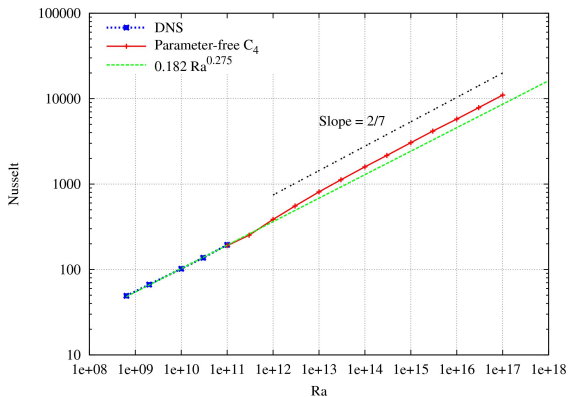
## Challenging $\mathcal{C}_4$ : mesh independence analysis



The overall Nusselt number and the centerline stratification for 50 randomly generated coarse grids with fixed stretching at  $Ra = 10^{10}$ .

$$8 \leq N_x \leq 16, 17 \leq N_y \leq 34, \text{ and } 40 \leq N_z \leq 80.$$

# Performance at very high Rayleigh numbers



Meshes have been generated with the criteria of keeping the same number of points in the BL than for  $Ra = 10^{10}$ .

## Conclusions and Future Research

The results shown illustrate the potential of the conservative parameter-free  $\tilde{\mathcal{C}}_4$  smoothing as a new simulation shortcut.

The main advantages with respect existing LES models can be summarized:

- **Robustness.** As the smoothed governing equations preserve the symmetry properties of the original NS equations the solution cannot blow up (in the energy-norm, in 2D also: enstrophy-norm). It seems that even for very coarse meshes reasonable results can be obtained.
- **Universality.** No *ad hoc* phenomenological arguments that cannot be formally derived for the NS equations are used.
- The proposed method constitutes a **parameter-free turbulence model**.

Thank you for you attention

## Further reading about $\mathcal{C}_4$ regularization

- Roel Verstappen, “*On restraining the production of small scales of motion in a turbulent channel flow*”, *Computers & Fluids*, 37 (7): 887-897, 2008
- F. X. Trias *et al.*, “*Parameter-free symmetry-preserving regularization modeling of a turbulent differentially heated cavity*”, *Computers & Fluids*, 39:1815-1831, 2010.
- F. X. Trias and R.W.C.P. Verstappen, “*On the construction of discrete filters for symmetry-preserving regularization models*”, *Computers & Fluids*, 40:139-148, 2011.