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Large Eddy Simulation for Aerodynamics and Aeroacoustics
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**Modelling and discretizing a turbulent
differentially heated cavity at $Ra = 10^{11}$**

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Presentation outline

1. Introduction

- Problem definition: Differentially Heated Cavity
- DNS results for $Ra = 10^{11}$, $Pr = 0.71$
- Governing equations

2. Regularization models for the simulation of turbulence

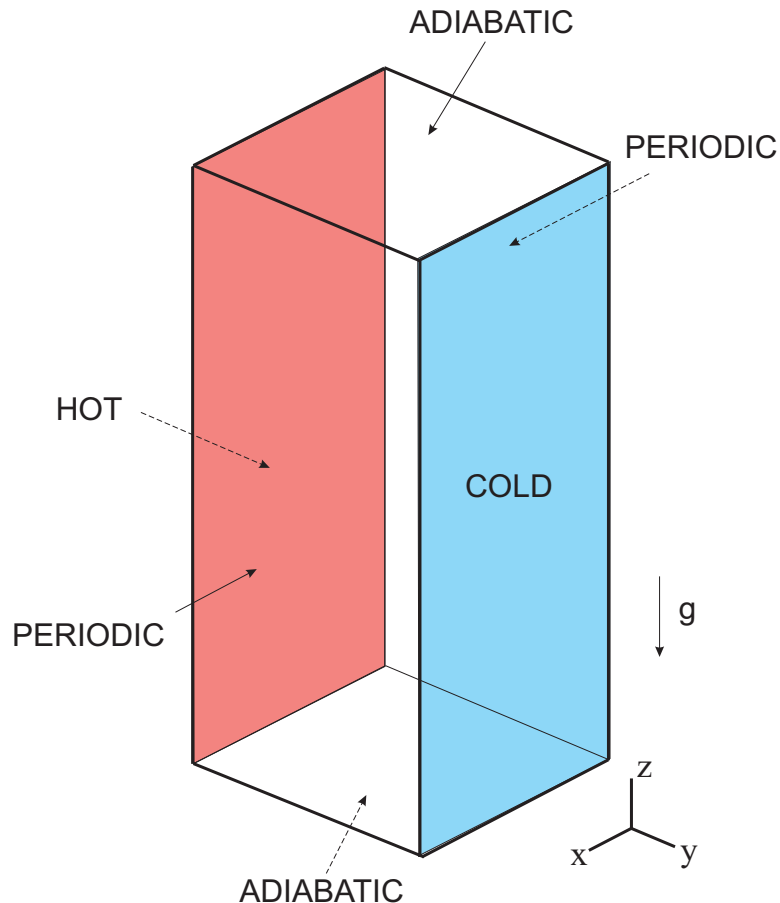
- Existing regularization: Leray and Navier-Stokes- α models
- Symmetry-preserving regularization models
- Mathematical foundation
- Discretizing the \mathcal{C}_n regularization modelling

3. Results for a Differentially Heated Cavity

- Description of cases
- Initial test: trial-and-error
- Parameter-free approach
 - ★ Comparison with convergence studies
 - ★ Mean fields

4. Conclusions and Future Research

Problem definition: Differentially Heated Cavity



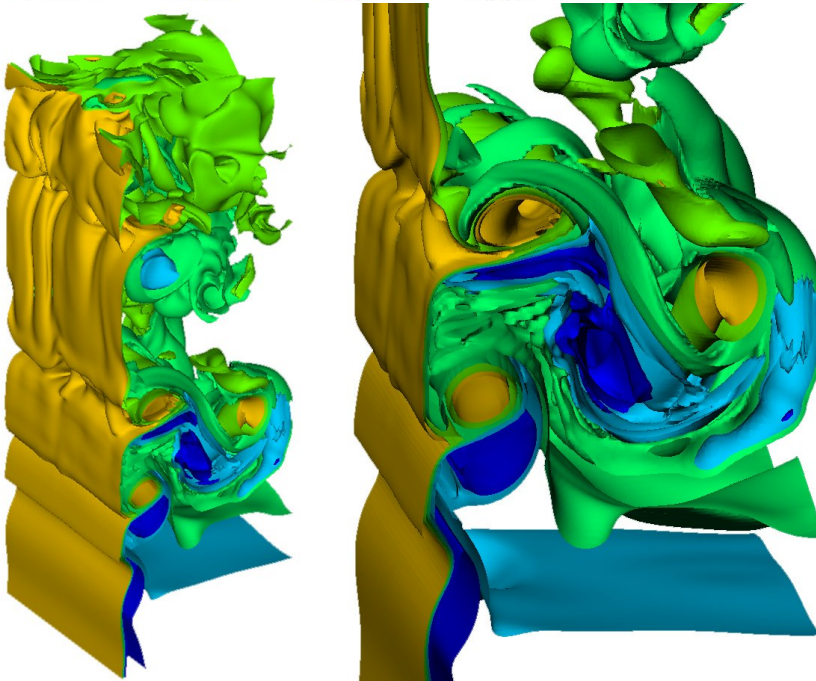
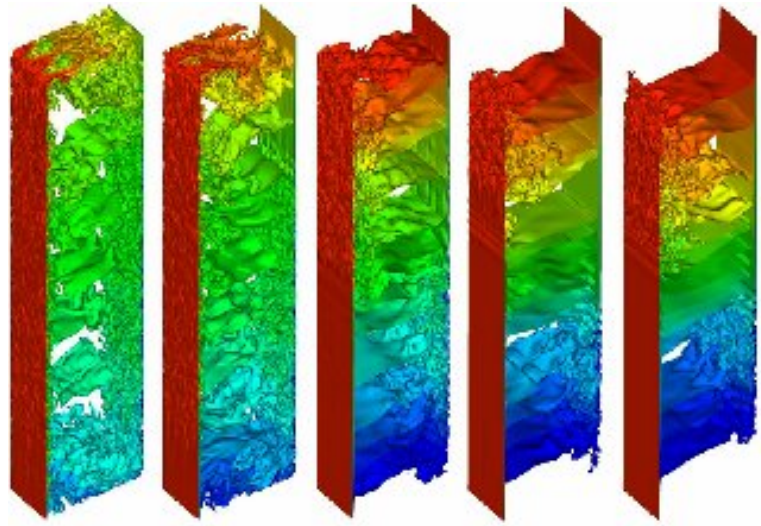
Boundary conditions:

- **Isothermal vertical walls**
- **Adiabatic horizontal walls**
- **Periodic** boundary conditions in the x -direction, orthogonal to the main flow

Dimensionless governing numbers:

- $Ra_z = \frac{\beta \Delta T L_z^3 g}{\alpha \nu}$
- $Pr = \frac{\nu}{\alpha}$
- Height aspect ratio $A_z = \frac{L_z}{L_y}$
- Depth aspect ratio $A_x = \frac{L_x}{L_y}$

DNS results for $Ra = 10^{11}$, $Pr = 0.71$



Some details about **DNS simulations**:

- Mesh size: $128 \times 682 \times 1278$
- Computing Time: ≈ 3 months - 256 CPUs
- 4^{th} -order symmetry-preserving discretization
- $A_z = 4$

Complexity of the flow:

- Boundary layers
- Stratified cavity core
- Internal waves
- Recirculation areas

Governing equations

Incompressible Navier-Stokes coupled with energy transport equation:

$$\begin{aligned}\nabla \cdot u &= 0 \\ \partial_t u + \mathcal{C}(u, u) &= Pr \mathcal{D}(u) - \nabla p + f \\ \partial_t T + \mathcal{C}(u, T) &= \mathcal{D}(T)\end{aligned}$$

where $f = (0, 0, RaPrT)$ (Boussinesq approximation) and the **nonlinear convective term** is given by

$$\mathcal{C}(u, v) = (u \cdot \nabla)v$$

and the linear dissipative term is given by

$$\mathcal{D}(u) = \frac{1}{Ra^{0.5}} \nabla^2 u$$

Regularization modelling

As the full energy spectrum cannot be computed, a **dynamically less complex mathematical formulation** is sought. We consider smooth approximations (regularizations) of the nonlinearity,

$$\begin{aligned}\partial_t u_\epsilon + \tilde{\mathcal{C}}(u_\epsilon, u_\epsilon) &= Pr\mathcal{D}(u_\epsilon) - \nabla p_\epsilon + f \\ \partial_t T_\epsilon + \tilde{\mathcal{C}}(u_\epsilon, T_\epsilon) &= \mathcal{D}(T_\epsilon)\end{aligned}$$

such approximations may fall in the **Large-Eddy Simulation** (LES) concept,

$$\begin{aligned}\partial_t \bar{u}_\epsilon + \mathcal{C}(\bar{u}_\epsilon, \bar{u}_\epsilon) &= Pr\mathcal{D}(\bar{u}_\epsilon) - \nabla \bar{p}_\epsilon + f + \mathcal{M}_1(\bar{u}_\epsilon, \bar{u}_\epsilon) \\ \partial_t \bar{T}_\epsilon + \mathcal{C}(\bar{u}_\epsilon, \bar{T}_\epsilon) &= \mathcal{D}(\bar{T}_\epsilon) + \mathcal{M}_2(\bar{u}_\epsilon, \bar{T}_\epsilon)\end{aligned}$$

if the filter is invertible:

$$\begin{aligned}\mathcal{M}_1(\bar{u}_\epsilon, \bar{u}_\epsilon) &= \mathcal{C}(\bar{u}_\epsilon, \bar{u}_\epsilon) - \overline{\tilde{\mathcal{C}}(u_\epsilon, u_\epsilon)} \\ \mathcal{M}_2(\bar{u}_\epsilon, \bar{T}_\epsilon) &= \mathcal{C}(\bar{u}_\epsilon, \bar{T}_\epsilon) - \overline{\tilde{\mathcal{C}}(u_\epsilon, T_\epsilon)}\end{aligned}$$

Previous regularization modellings

Leray and Navier-Stokes- α models

The regularization methods basically **alters the convective term** to **restrain the production of small scales** of motion.

- Leray model:

$$\partial_t u_\epsilon + \mathcal{C}(\bar{u}_\epsilon, u_\epsilon) = \mathcal{D}(u_\epsilon) - \nabla p_\epsilon$$

- Navier-Stokes- α model:

$$\partial_t u_\epsilon + \mathcal{C}_r(u_\epsilon, \bar{u}_\epsilon) = \mathcal{D}(u_\epsilon) - \nabla \pi_\epsilon$$

where the $\pi = p + u^2/2$ and the convective operator in rotational form is defined as

$$\mathcal{C}_r(u, v) = (\nabla \times u) \times v$$

However, in doing so some of the **inviscid invariants** (kinetic energy, enstrophy in 2D and helicity in 3D) are **not conserved**.

Symmetry-preserving regularization models (1/2)

In order to conserve the following inviscid invariants

- Kinetic energy

$$\int_{\Omega} \mathbf{u} \cdot \mathbf{u} d\Omega$$

- Enstrophy (in 2D)

$$\int_{\Omega} (\nabla \times \mathbf{u}) \cdot (\nabla \times \mathbf{u}) d\Omega$$

- Helicity (in 3D)

$$\int_{\Omega} (\nabla \times \mathbf{u}) \cdot \mathbf{u} d\Omega$$

the **approximate convective operator** has to be **skew-symmetric**:

$$\left(\tilde{\mathcal{C}}(u, v), w \right) = - \left(\tilde{\mathcal{C}}(u, w), v \right)$$

Symmetry-preserving regularization models (2/2)

This criterion yields the following class of approximations,

$$\partial_t u_\epsilon + \mathcal{C}_n(u_\epsilon, u_\epsilon) = \mathcal{D}(u_\epsilon) - \nabla p_\epsilon$$

in which the convective term is smoothened according to:

$$\mathcal{C}_2(u, v) = \overline{\mathcal{C}(\bar{u}, \bar{v})}$$

$$\mathcal{C}_4(u, v) = \mathcal{C}(\bar{u}, \bar{v}) + \overline{\mathcal{C}(\bar{u}, v')} + \overline{\mathcal{C}(u', \bar{v})}$$

$$\mathcal{C}_6(u, v) = \mathcal{C}(\bar{u}, \bar{v}) + \mathcal{C}(\bar{u}, v') + \mathcal{C}(u', \bar{v}) + \overline{\mathcal{C}(u', v')}$$

where $u' = u - \bar{u}$ and $\mathcal{C}_n(u, v) = \mathcal{C}(u, v) + \mathcal{O}(\epsilon^n)$ for **any symmetric filter**.

Mathematical foundation

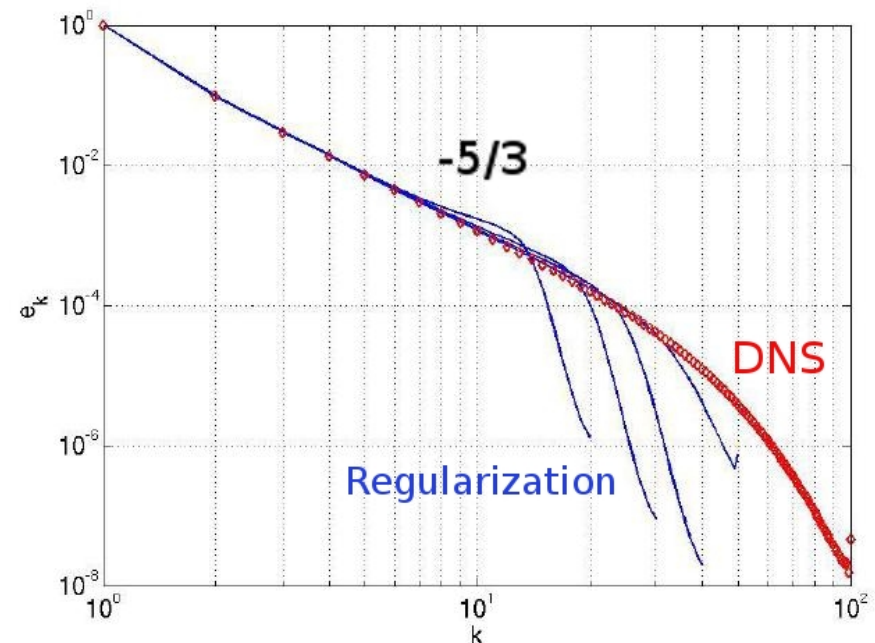
Energy flux equation for the symmetry-preserving regularization resembles the NS

$$\frac{1}{2} \frac{d}{dt} |u_{kk'}|^2 + \nu |\nabla u_{kk'}|^2 = \tilde{T}_k - \tilde{T}_{k'} \quad \longrightarrow \quad \nu \langle |\nabla u_{kk'}|^2 \rangle = \langle \tilde{T}_k \rangle - \langle \tilde{T}_{k'} \rangle$$

\implies Following the same steps as Foias *et al.* (2001)

- $\langle \tilde{T}_k \rangle$ is a nonnegative, monotone decreasing function.
- $\langle \tilde{T}_k \rangle$ is approximately constant for $k_a < k < k_b$ (existence of inertial range).

\implies **$-5/3$ scaling !!!**



LES-interpretation of \mathcal{C}_4 -regularization

$$\partial_t \bar{u}_\epsilon + \mathcal{C}(\bar{u}_\epsilon, \bar{u}_\epsilon) - \mathcal{D}(\bar{u}_\epsilon) + \nabla \bar{p}_\epsilon =$$

$$\mathcal{C}(\bar{u}_\epsilon, \bar{u}_\epsilon) - \overline{\mathcal{C}_4(u_\epsilon, u_\epsilon)} =$$

$$-\frac{\epsilon^2}{12} \nabla \cdot (\nabla \bar{u}_\epsilon : \nabla \bar{u}_\epsilon) + \mathcal{O}(\epsilon^4)$$

gradient model + stabilization

Discretizing the \mathcal{C}_n regularization modelling

- The discretization is also a regularization. The **spatial discretization** method preserves the symmetry and conservation properties too

$$\Omega_s \frac{d\mathbf{u}_s}{dt} + \mathcal{C}(\mathbf{u}_s) \mathbf{u}_s + D \mathbf{u}_s + \Omega_s G \mathbf{p}_c = \mathbf{0}_s \quad \text{with} \quad \mathcal{C}(\mathbf{u}_s) = -\mathcal{C}^*(\mathbf{u}_s)$$

and is therefore well-suited to test the proposed regularization model.

- A normalized self-adjoint **filter** has been chosen. In 1D it becomes

$$\bar{\phi}_i = \frac{\epsilon^4 - 4\epsilon^2}{1152} (\phi_{i+2} + \phi_{i-2}) + \frac{16\epsilon^2 - \epsilon^4}{288} (\phi_{i+1} + \phi_{i-1}) + \frac{\epsilon^4 - 20\epsilon^2 + 192}{192} \phi_i$$

Results for differentially heated cavity at $Ra = 10^{11}$

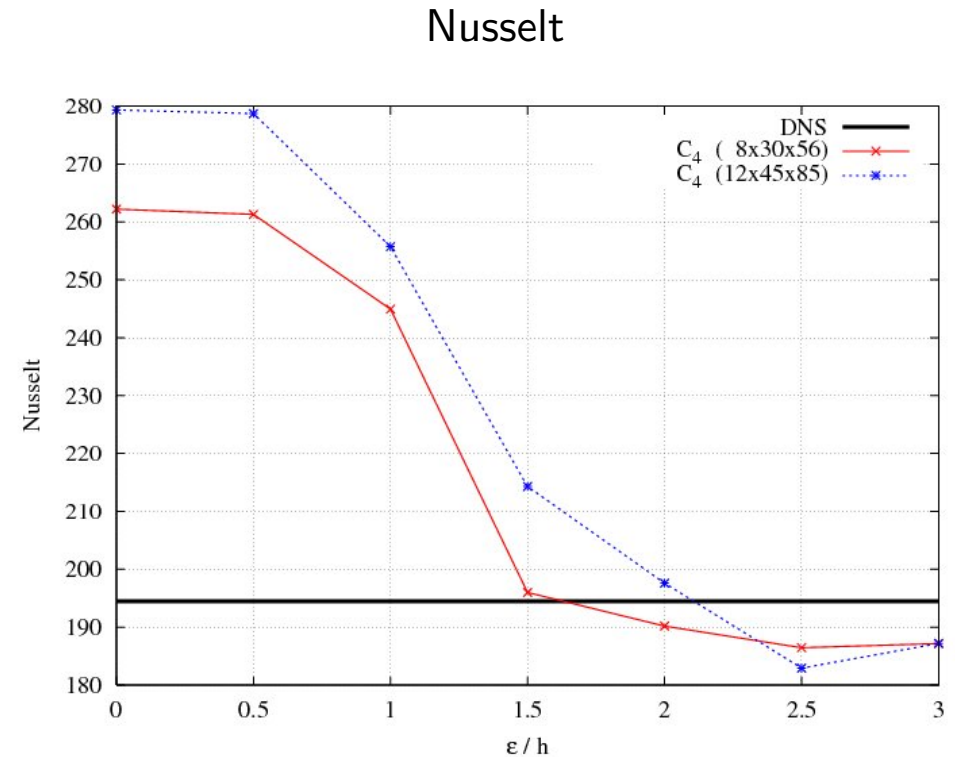
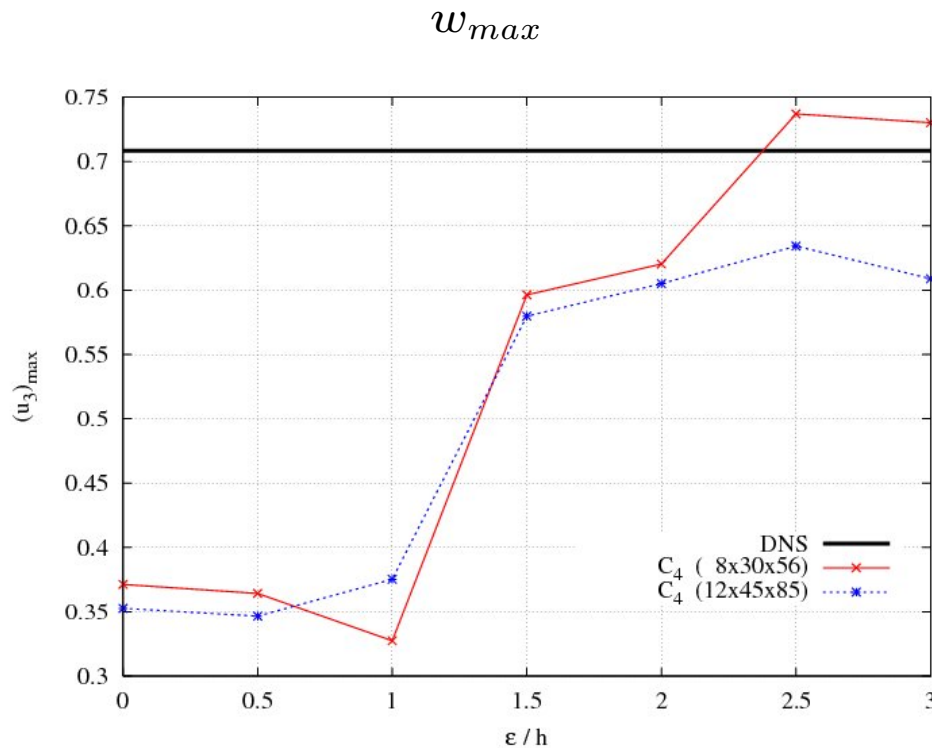
- Regularization model \mathcal{C}_4 is tested.
- Two coarse meshes are considered

	DNS	RM1	RM2
N_x	128	12	8
N_y	682	45	30
N_z	1278	85	56
Δx_{min}	3.79×10^{-3}	4.16×10^{-2}	6.25×10^{-2}
Δy_{min}	2.16×10^{-4}	3.27×10^{-3}	4.91×10^{-3}
Δz_{min}	3.13×10^{-3}	4.71×10^{-2}	7.14×10^{-2}

- Initial test:** ratio ϵ/h (filter length to the average grid width) is kept constant in all three spatial directions.

Results for differentially heated cavity at $Ra = 10^{11}$

Convergence studies



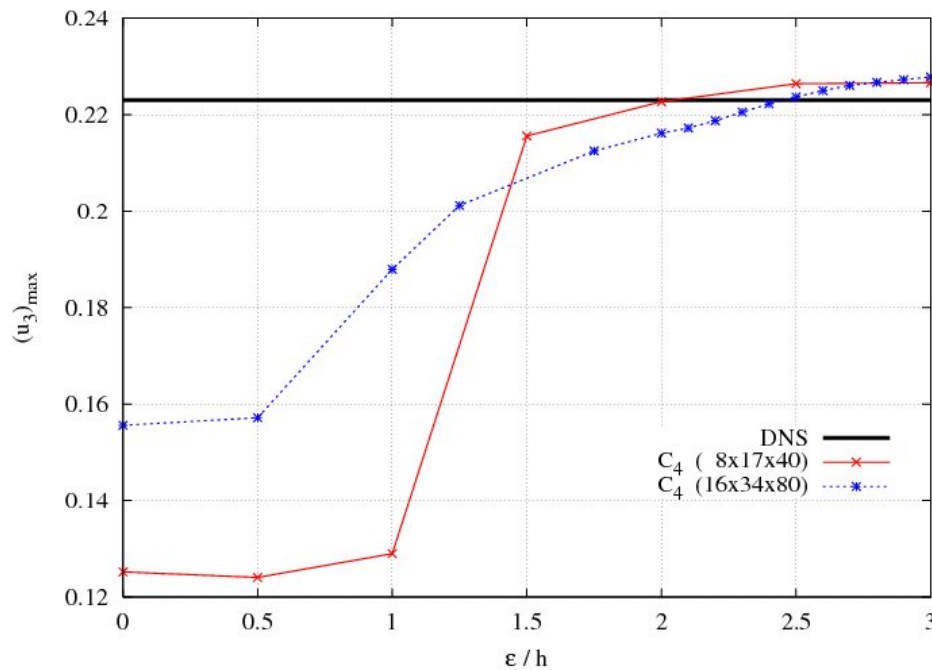
The maximum of the averaged vertical velocity at the horizontal mid-height plane and the overall averaged Nusselt number as a function of the ratio of the filter length ϵ to the average grid width h .

⇒ A **weak dependance** for sufficiently **large values of ϵ** is observed

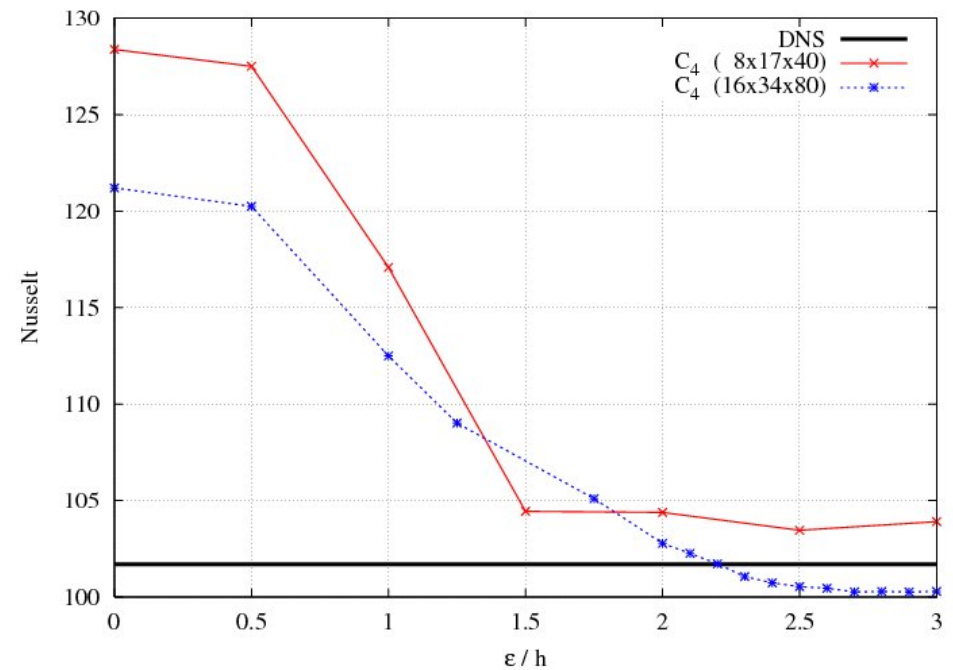
Same behaviour has also been observed at different Ra -numbers!

Convergence studies at $Ra = 10^{10}$

w_{max}



Nusselt



The maximum of the averaged vertical velocity at the horizontal mid-height plane and the overall averaged Nusselt number as a function of the ratio of the filter length ϵ to the average grid width h .

\Rightarrow A **weak dependance** for sufficiently **large values of ϵ** is observed

Parameter-free approach

The **convective and diffusive contributions** to the evolution of $|v|^2$ are given by

$$\frac{v \cdot \mathcal{C}(u, v)}{v \cdot v} = \frac{v \cdot \mathcal{S}(u) v}{v \cdot v} \quad \text{and} \quad \frac{v \cdot \mathcal{D}v}{v \cdot v} = \nu \frac{\nabla v : \nabla v}{v \cdot v}$$

At the **smallest grid scale**, $k = \pi/h$, **convection dominates** diffusion

$$\frac{v_k \cdot \mathcal{C}(u, v)_k}{v_k \cdot v_k} > \nu k^2$$

\implies In the present work we **determine the filter width** ϵ from

$$\frac{v_k \cdot \mathcal{C}_4(u, v)_k}{v_k \cdot v_k} \approx \nu k^2$$

Parameter-free approach

Note that $\mathcal{C}_4(u, v)$ depends on the filter length ϵ . For the **smallest scale** this dependence becomes

$$\frac{v_k \cdot \mathcal{C}_4(u, v)_k}{v_k \cdot v_k} \approx f_4(\hat{g}_k(\epsilon)) \frac{v_k \cdot \mathcal{S}(u) v}{v_k \cdot v_k} \leq f_4(\hat{g}_k(\epsilon)) \lambda_{max}(\mathcal{S})$$

where $0 < \hat{g}_k(\epsilon) \leq 1$ is the transfer function of the filter and the damping function $0 < f_4 \leq 1$.

\implies Therefore, it suffices that following inequality be **locally** hold

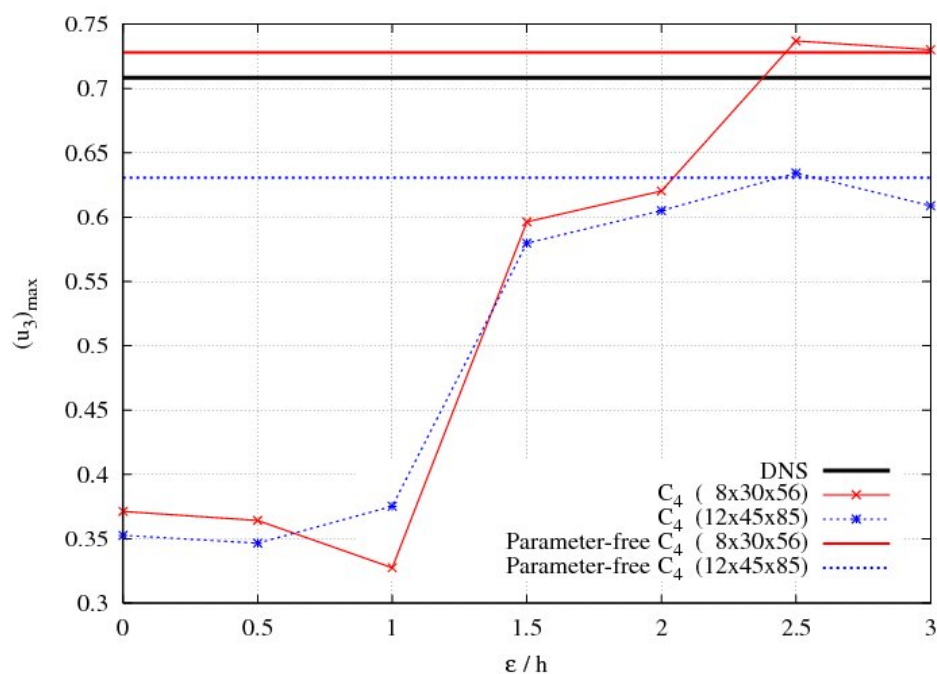
$$f_4(\hat{g}_k(\epsilon)) \leq \frac{\nu k^2}{\lambda_{max}(\mathcal{S})} \longrightarrow \epsilon$$

to guarantee that the **production of smaller scales of motion be stopped at the smallest scale** set by the mesh.

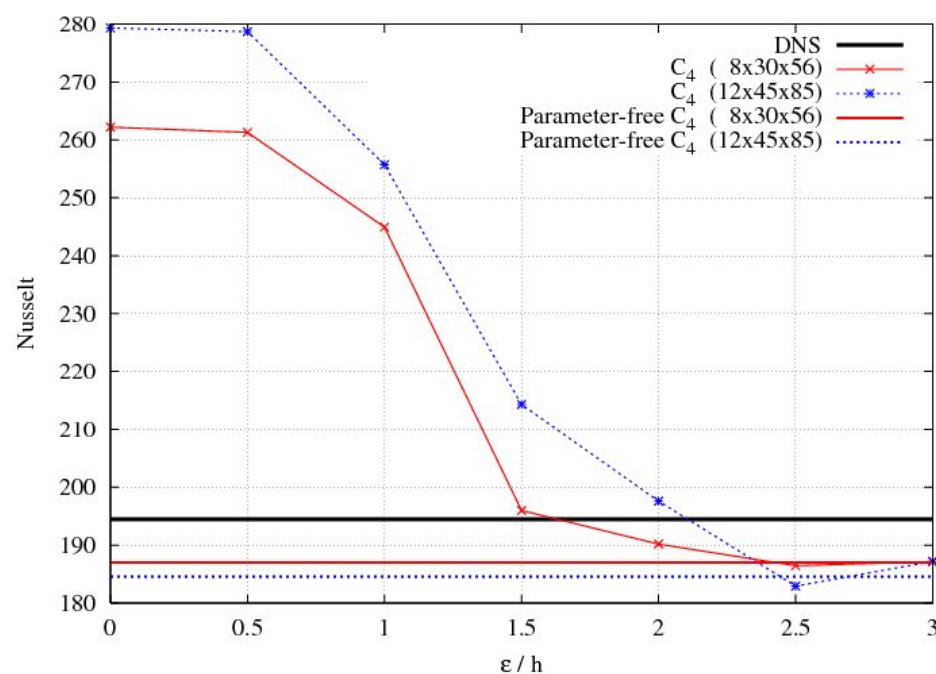
Results for differentially heated cavity at $Ra = 10^{11}$

Free-parameter approach

w_{max}



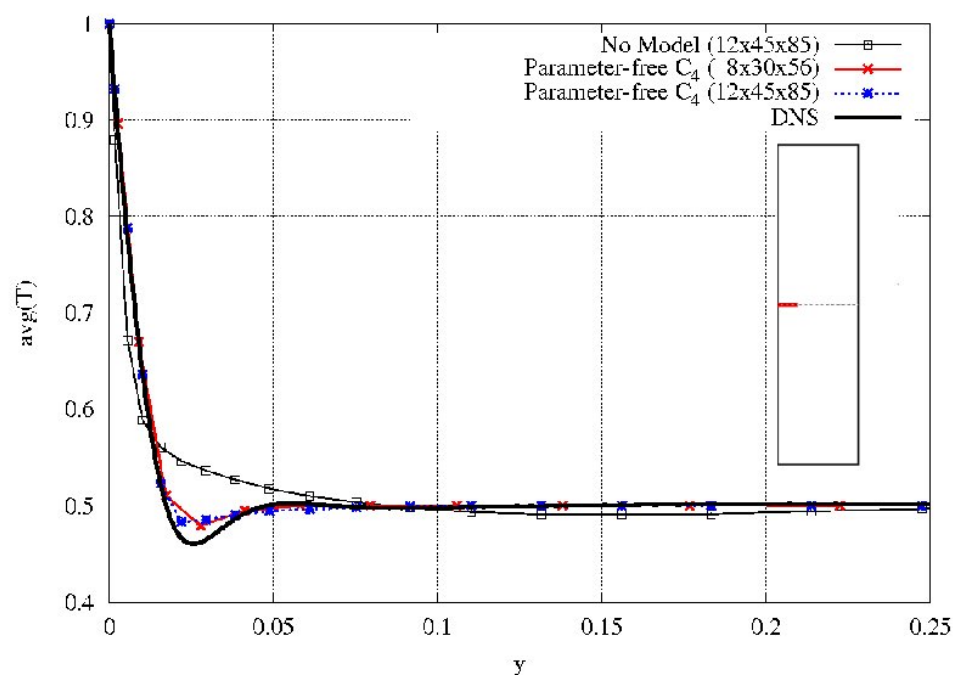
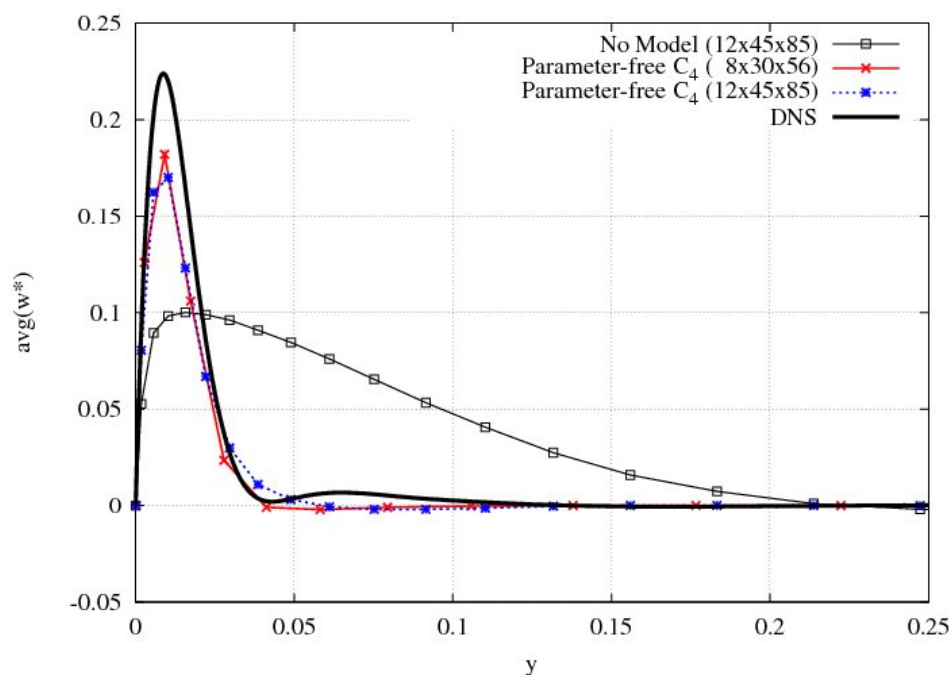
Nusselt



The maximum of the averaged vertical velocity at the horizontal mid-height plane and the overall averaged Nusselt number as a function of the ratio of the filter length ϵ to the average grid width h .

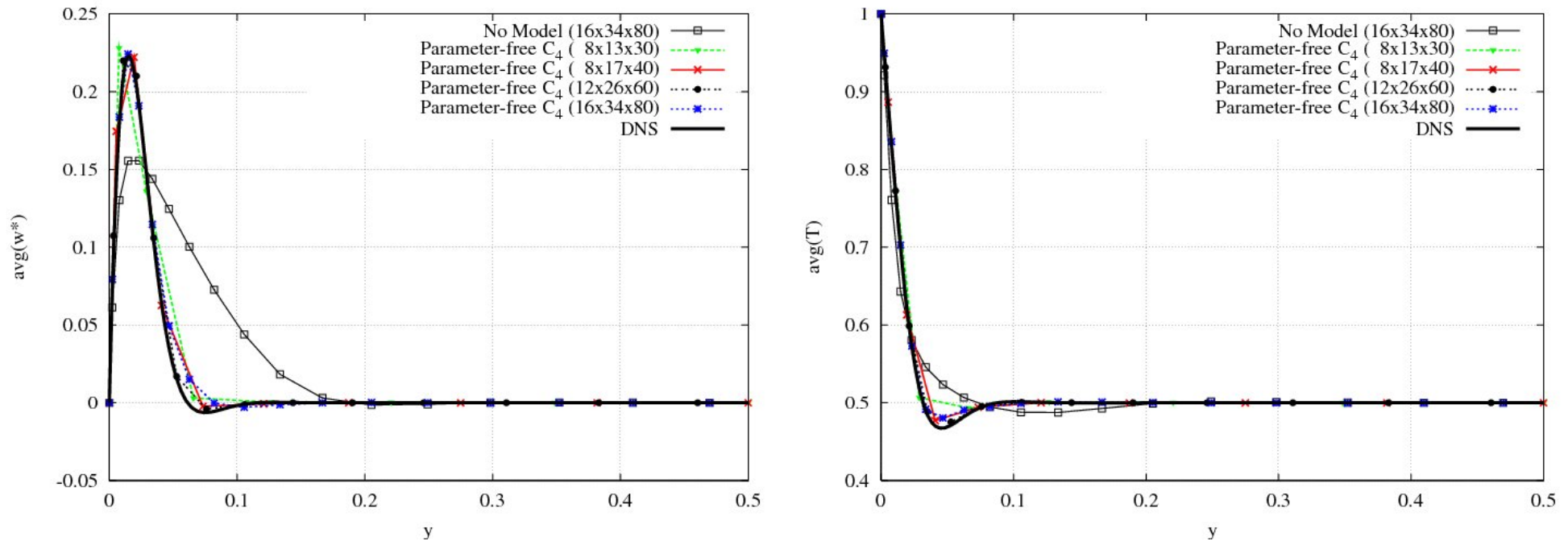
Results for differentially heated cavity at $Ra = 10^{11}$

Profiles



Averaged vertical velocity and temperature profiles at the horizontal mid-height plane.

How does the parameter-free \tilde{C}_4 symmetry-preserving regularization modelling behave for other grids and Ra -numbers?



Averaged vertical velocity and temperature profiles at the horizontal mid-height plane at $Ra = 10^{10}$.

Even for a **very coarse** $8 \times 13 \times 30$ grid **reasonable results** are being obtained!

\Rightarrow Results for different grids show the **robustness** of the method.

Conclusions and Future Research

The results shown illustrate the potential of the conservative parameter-free $\tilde{\mathcal{C}}_4$ smoothing as a new simulation shortcut.

The main advantages with respect exiting LES models can be summarized:

- **Robustnest.** As the smoothed governing equations preserve the symmetry properties of the original Navier-Stokes equations the solution cannot blow up (in the energy-norm, in 2D also: enstrophy-norm). It seems that even for very coarse meshes reasonably results can be obtained.
- **Universality.** No *ad hoc* phenomenological arguments that can not be formally derived for the Navier-Stokes equations are used.
- The proposed method constitutes a **parameter-free turbulence model**.

Since now, the method has been **successfully tested** on completely different turbulent configurations such as:

- Channel flow.
- Impinging jet.
- **Differentially heated cavity** at different Ra -numbers.

Thank you for you attention

Discretization of the convective operator: a symmetry-preserving discretization

The spatially discrete incompressible Navier-Stokes equations are expressed as

$$\Omega_s \frac{d\mathbf{u}_s}{dt} + \mathbf{C}(\mathbf{u}_s) \mathbf{u}_s + \mathbf{D} \mathbf{u}_s + \Omega_s \mathbf{G} \mathbf{p}_c = \mathbf{0}_s$$
$$\mathbf{M} \mathbf{u}_s = \mathbf{0}_c$$

⇒ It was shown that the **convective matrix** $\mathbf{C}(\mathbf{u}_s)$ has to be **skew-symmetric**,

$$\mathbf{C}(\mathbf{u}_s) + \mathbf{C}^*(\mathbf{u}_s) = \mathbf{0}$$

... to **preserve** the continuous **invariants** (kinetic energy, enstrophy in 2D and helicity in 3D) in a **discrete sense**.

Choice of the filter

Let us consider a generic linear filter

$$\bar{u}_\epsilon = F u_\epsilon$$

Then, three basic properties are required for the filter:

$$\begin{aligned}\bar{u}_\epsilon &= u_\epsilon + \mathcal{O}(\epsilon^2) \\ (\Omega_s F) &= (\Omega_s F)^* \\ F 1 &= 1\end{aligned}$$

\implies Our discrete filter is a **5-point Gaussian filter**.